

MODULE -5 REFRIGERATION

- ☐ Air refrigeration system
- ☐ Vapour compression refrigeration system - Components - Working - P-H and T-S diagrams - Calculation of COP - Effect of sub-cooling and super-heating
- ☐ Selection and properties of refrigerant - Vapour absorption system - NH₃ – water system,
- ☐ Vapour adsorption system.
- ☐ Cryogenic engineering - Introduction, Application, Cryo-coolers.

Numerical on VCR System

1. A refrigerator works between -7°C and 27°C . The vapour is dry at the end of adiabatic compression. There is no under cooling. Determine the following,
- (i) C.O.P (ii) Power of compressor to remove 12140 kJ/h of heat.
(iii) capacity of the system

Temp ($^{\circ}\text{C}$)	Sensible heat h_f (kJ/kg)	Latent heat h_{fg} (kJ/kg)	Entropy of liquid (s_f) (kJ/kg-K)	Entropy of vapour (s_g) (kJ/kg-K)
-7	-29.3	1297.9	-0.109	4.748
27	117.23	1172.3	0.427	4.333

Given data :

$$R.E. = 12140 \text{ kJ} / \text{hr} = \frac{12140}{3600} \text{ kJ} / \text{s} = 3.37 \text{ kW}$$

Numerical on VCR System

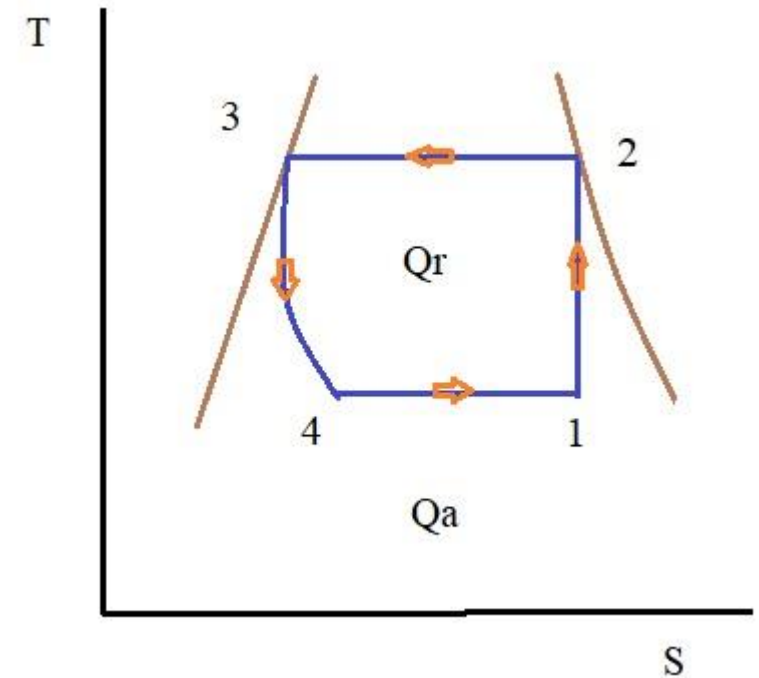
Temp (°C)	Sensible heat h_f (kJ/kg)	Latent heat h_{fg} (kJ/kg)	Entropy of liquid (s_f) (kJ/kg-K)	Entropy of vapour (s_g) (kJ/kg-K)
-7	-29.3	1297.9	-0.109	4.748
27	117.23	1172.3	0.427	4.333

To Find :

$$(i) C.O.P = \frac{h_1 - h_4}{h_2 - h_1}$$

$$(ii) \text{Power of compressor} = \dot{m}_r \times (h_2 - h_1)$$

$$(iii) \text{capacity of the system} = \frac{R.E \text{ in kW}}{3.5 \text{ kW}} (TR)$$



Numerical on VCR System

Solution :

$h_1 \dots ?$

$$h_1 = h_{f1} + x_1 h_{fg1}$$

$$s_1 = s_{f1} + x_1 s_{fg1} = s_{f1} + x_1 (s_{g1} - s_{f1})$$

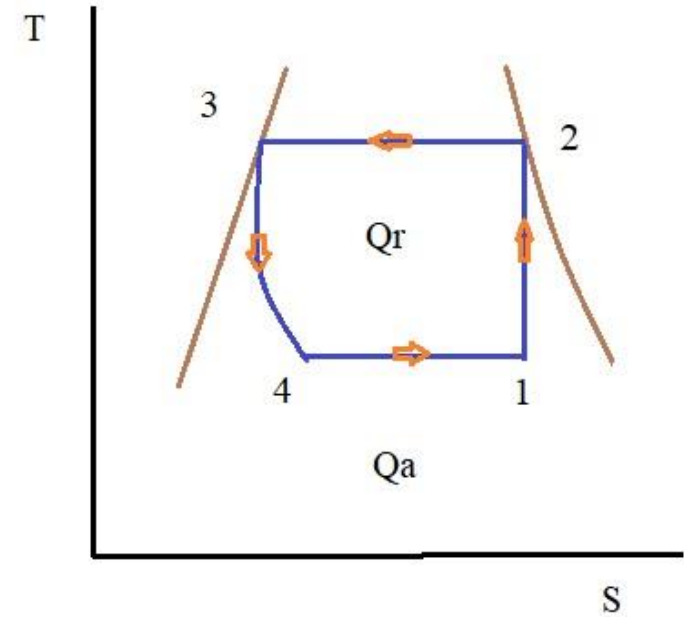
$$\therefore s_2 = s_{g2} = 4.333 (\text{kJ} / \text{kg} \cdot \text{K}) = s_1$$

$$4.333 = -0.109 + x_1 (4.748 + 0.109)$$

$$x_1 = 0.914$$

$$h_1 = h_{f1} + x_1 h_{fg1} = -29.3 + 0.914(1297.9)$$

$$h_1 = 1157.7 \text{ kJ} / \text{kg}$$



Numerical on VCR System

Find : $h_2 \dots ?$

$$h_2 = h_{f2} + x_2 h_{fg2} = 117.23 + 1172.3 = 1289.63 \text{ (kJ / kg)}$$

Find : $h_3 = h_{f3} = 117.23 \text{ (kJ / kg)}$

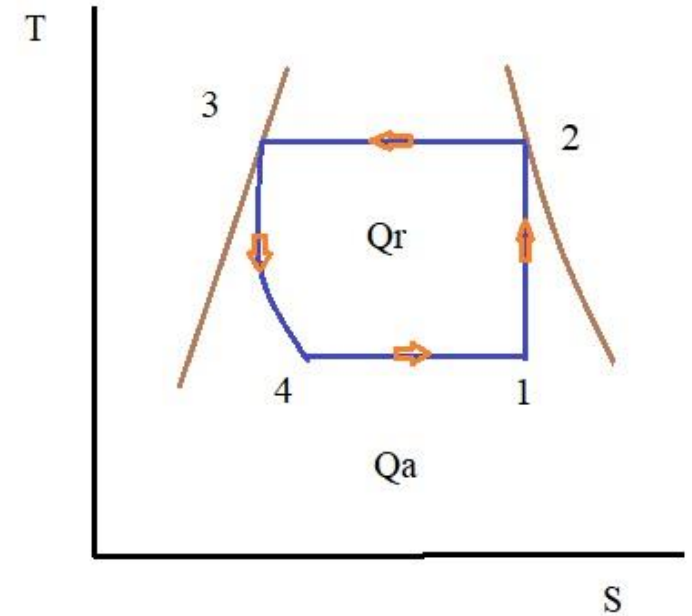
Answers :

$$(i) COP = \frac{h_1 - h_3}{h_2 - h_1} = 8$$

$$(ii) \text{Power of compressor} = \dot{m}_r \times (h_2 - h_1) = 0.0032 \times (1289.6 - 1157.7) = 0.424 \text{ kW}$$

$$\text{Where, } \dot{m}_r = \frac{12140 / 3600}{(h_1 - h_4)} = 0.0032 \text{ kg / s}$$

$$(iii) \text{Capacity of the system} = \frac{R.E. \text{ in kW}}{3.5 \text{ kW}} = \frac{12140 / 3600}{3.5} = 0.96 \text{ T.R.}$$



Numerical on VCR System

2. A VCR system uses R-12 as a refrigerant. The liquid refrigerant evaporates in the evaporator at -15°C and the vapour is condensed in condenser at 10°C . The temperature of the refrigerant at the delivery of the compressor is 15°C . Determine the COP (i) if there is no under cooling, (ii) if the refrigerant is under cooled 5°C before throttling.

Take $C_{pv} = 0.658 \text{ kJ/kg-K}$ and $C_{pL} = 0.948 \text{ kJ/kg-K}$.

Properties of R-12:

Temp ($^{\circ}\text{C}$)	Sensible heat h_f (kJ/kg)	Enthalpy of Vapour h_g (kJ/kg)	Entropy of liquid (s_f) (kJ/kg-K)	Entropy of vapour (s_g) (kJ/kg-K)
-15	185.32	345.39	0.9450	1.5675
10	209.41	357.05	1.0335	1.555

Numerical on VCR System

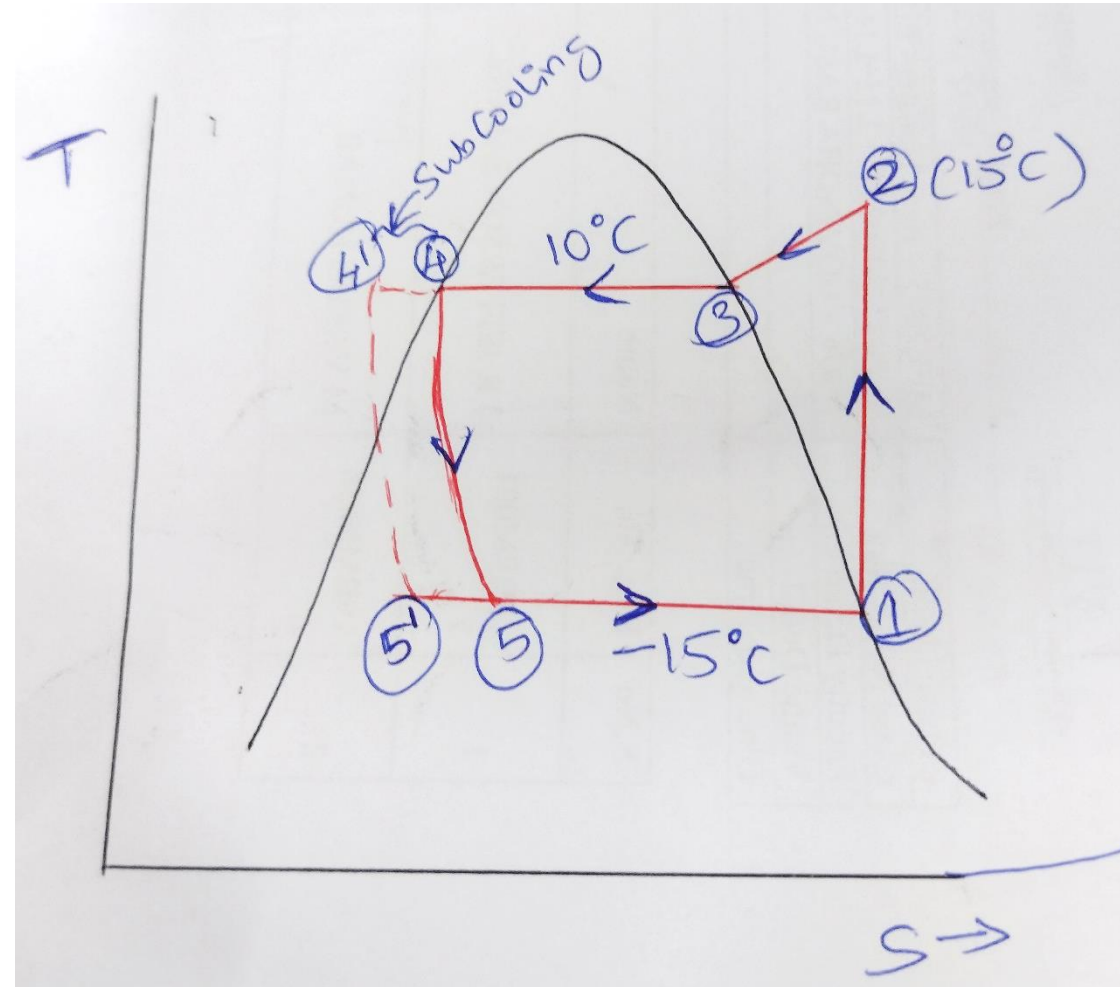
To Find :

(i) COP (without under – cooling or sub – cooling)

$$= \frac{h_1 - h_5}{h_2 - h_1} = \frac{h_1 - h_4}{h_2 - h_1}$$

(ii) COP (with undercooling or sub – cooling)

$$= \frac{h_1 - h_5^1}{h_2 - h_1} = \frac{h_1 - h_4^1}{h_2 - h_1}$$



Numerical on VCR System

$$(i) COP (\text{without undercooling}) = \frac{h_1 - h_4}{h_2 - h_1}$$

State - 1 @ -15°C

$$h_1 = h_f + x_1 h_{fg1}$$

$$s_1 = s_f + x_1 s_{fg1} \Rightarrow x_1 ?$$

here,

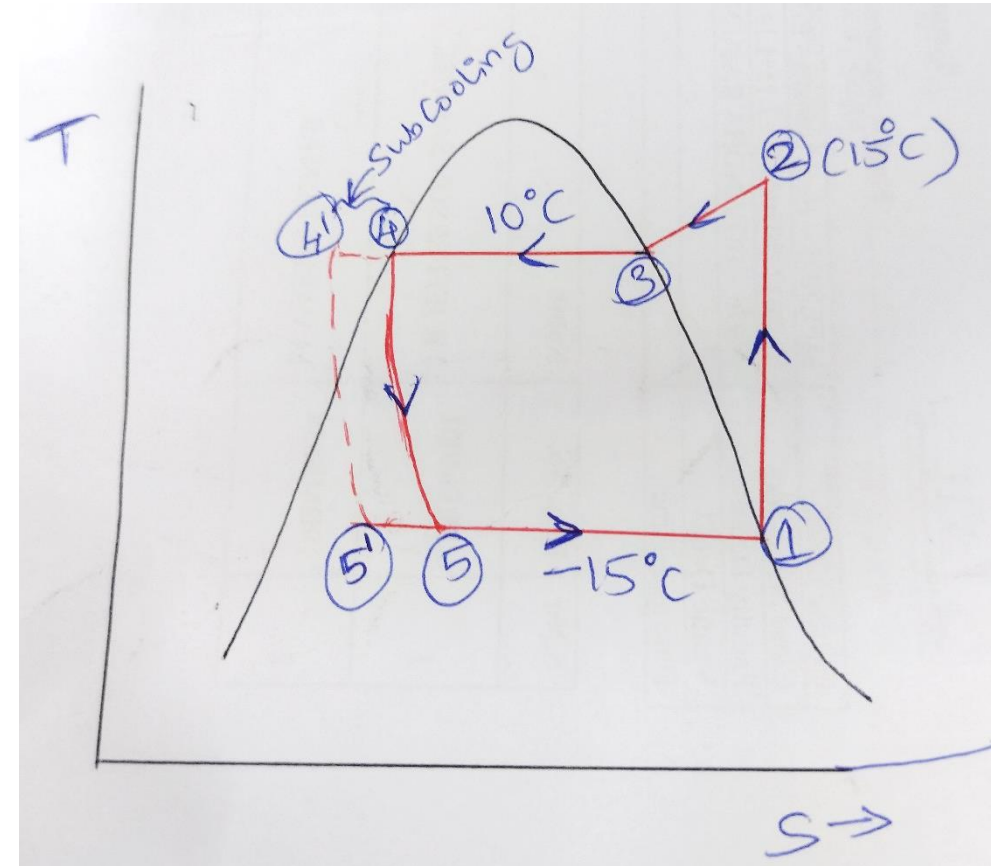
$$h_{fg} = h_g - h_f$$

$$s_{fg} = s_g - s_f$$

State - 2 @ 10°C

$$h_2 = h_{g2} + c_{pv} (T_{\text{sup}} - T_{\text{sat}})$$

$$s_2 = s_{g2} + c_{pv} \ln \left(\frac{T_{\text{sup}}}{T_{\text{sat}}} \right) = s_1$$



Numerical on VCR System

State – 2 @ 10° C

$$h_2 = h_{g_2} + c_{pv}(T_{\text{sup}} - T_{\text{sat}}) = 357.05 + 0.658(288 - 283) = 360.29 \text{ kJ / kg}$$

$$s_2 = s_{g_2} + c_{pv} \ln \left(\frac{T_{\text{sup}}}{T_{\text{sat}}} \right) = 1.555 + 0.658 \ln \left(\frac{288}{283} \right) = 1.56 \text{ (kJ / kg.K)}$$

State – 1 @ –15° C (∵ s₁ = s₂)

$$s_1 = s_{f1} + x_1 s_{fg1} \Rightarrow 1.566 = 0.945 + x_1 (1.5675 - 0.945) \Rightarrow x_1 = 0.997$$

$$h_1 = h_{f1} + x_1 h_{fg1} = 185.32 + 0.997(345.39 - 185.32) = 344.9 \text{ kJ / kg}$$

$$h_4 = h_{f@10^\circ \text{C}} = 209.41 \text{ kJ / kg} = h_5$$

$$(i) \text{COP (without undercooling)} = \frac{h_1 - h_5}{h_2 - h_1} = \frac{h_1 - h_4}{h_2 - h_1} = 8.41$$

Numerical on VCR System

$$(ii) COP (with under cooling) = \frac{h_1 - h_5^1}{h_2 - h_1} = \frac{h_1 - h_4^1}{h_2 - h_1}$$

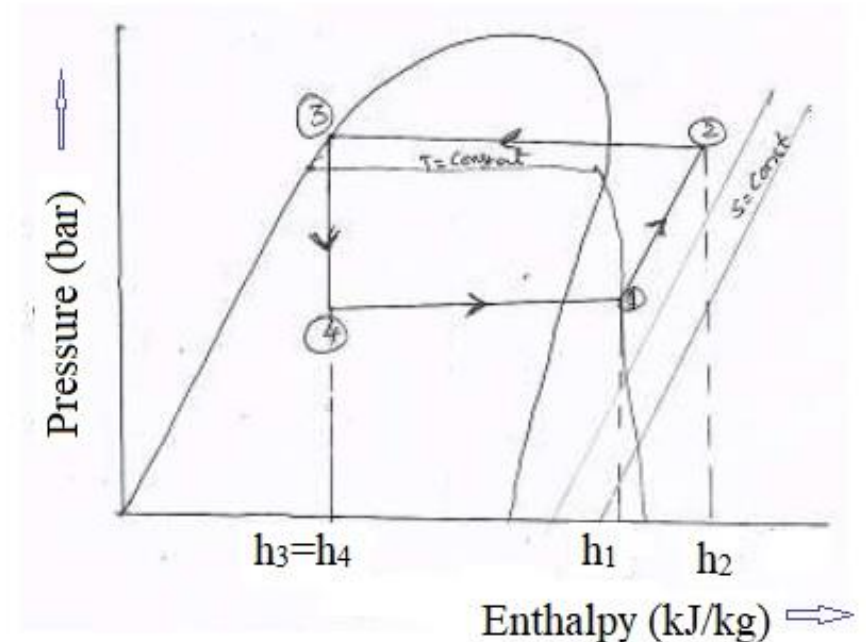
$$h_4^1 = h_{f4} - c_{pL} (T_{sat} - T_{sub}) = 209.41 - 0.948(5) = 204.67 \text{ (kJ / kg)}$$

$$COP = \left(\frac{344 - 204.67}{360 - 344} \right) = 8.7$$

Numerical on VCR System

3. A vapour compression refrigeration system uses R22 refrigerant as refrigerant. It is working between the pressure limits of 3 bar and 20 bar. Refrigerant temperatures are at the outlet of the evaporator 8°C , at the exit of the compressor 83°C , at the exit of the condenser 40°C and at the exit of the expansion valve 3°C . The condenser water inlet temperature is 18.8°C , and the outlet temperature is 38°C . The water flow rate to the condenser is $100\text{ cm}^3/\text{s}$. The power required to drive the compressor is 2.6 kW . Calculate the following: (using p-h chart)

- (i) Mass flow of refrigerant
- (ii) Refrigerating capacity
- (iii) Heat rejected by condenser
- (iv) Actual COP of the system
- (v) Carnot COP



Numerical on VCR System

Formula :

$$(i) m_r = \frac{W_{compressor}}{h_2 - h_1} (kg / s)$$

$$(ii) \text{Cooling capacity} = \frac{m_r (h_1 - h_4)}{3.5} (\text{in TR})$$

$$(iii) Q_{condenser} = m_w c_{pw} (T_{out} - T_{in})$$

$$(iv) \text{Actual COP} = \frac{h_1 - h_4}{h_2 - h_1}$$

$$(v) \text{Carnot COP} = \frac{T_1}{T_2 - T_1}$$

Numerical on VCR System

Solution:

$$h_1 = 410 \text{ kJ/kg}$$

$$h_2 = 460 \text{ kJ/kg}$$

$$h_3 = h_4 = 240 \text{ kJ/kg}$$

$$\text{Mass flow rate of the refrigerant, } m_r = W / (h_2 - h_1) = 0.0523 \text{ kg/sec}$$

$$\text{Refrigerating capacity} = m_r (h_1 - h_4) / 3.5167 = 2.53 \text{ TR}$$

$$\text{Heat rejected by the condenser, } Q_r = m_w C_{pw} (T_5 - T_6) = 8.02 \text{ kW}$$

$$\text{Actual COP} = \text{Refrigeration effect} / \text{Work done} = (h_1 - h_4) / (h_2 - h_1) = 3.4$$

$$\text{Carnot COP} = \frac{T_1}{T_2 - T_1} = 3.71$$